AUTOMATIC REGULATION AND OPTIMAL CONTROL REGARDING FLUVIAL OR SPATIAL NAVES EQUILIBRIUM STABILIZATION

Mircea LUPU¹ Gheorghe RADU² Cristian-George CONSTANTINESCU³

¹Prof., PhD, "Transilvania" University of Brasov, member of Romanian Scientists Academy
 ²Assist. Prof., PhD, "Henri Coanda" Air Force Academy, Brasov
 ³Assist. Prof. eng., PhD, "Henri Coanda" Air Force Academy, Brasov

Dedicated to math. Gheorghe Radu, PhD, on his 65-th anniversary

Abstract: The first part of the paper deals with cruise, cargo or underwater naves equilibrium stabilization in case of rolling perturbations. The stabilization conditions are determined by using a hydro-pneumatic automate regulator. Oscillations damping is achieved with a hydro-pneumatic compensator, by using the water tanks that the naves are equipped with.

The second part of the paper deals with automatic stabilization of rockets, submarines or satellites dynamics. This stabilization is based on relay-type automatic regulators, by using the minimal time criterion for optimal control with the Pontreagiune extremal principle. In this study, the state variables are the rotation angles and the control function has 2 components, which are appearing because of lateral rolling perturbations. Finaly, numeric-analytical I studies are approached, and the results are graphically presented.

Key words: optimal control, control function, extreme principle of Pontreaguine, absolute stability. *MSC2010:* 34H05, 49K35, 93C15, 93C73, 93D10.

1. INTRODUCTION

The goal of an optimal control problem (O.C.P.) of a dynamic system is to determine a set of state variables, of certain control and driving functions satisfying an optimization criterion, performing the extremization of aq quality index in this way. This performance index is a functional depending on these elements and time and spatial restrictions... Practical applications requirements for these functionals are optimal controls of the following type: achievement of minimal time, minimum fuel consumption, energy, to achieve extreme performance [4,10,13]. Dynamical systems from different domains are generally represented generally by nonlinear equations with parameters, while internal or external disturbances occur leading to unstable solutions related to a free balance state. The stabilization of these regimes is done by using automatic controls that actually fast reacts for optimal control and routing [3,5-7, 12,14]. Lurie [8], [4], [13] and Popov [11], [4], [3] methods are known to automatically adjust the absolute stabilization, with applications. This paper, for optimal control of stabilizing angular

velocities of aircraft and missiles, deals with the "Minimum Time Criteria" and the Pontreaguine extremal [9], [1], [7], with results and studies in different applications [5][6][7]. The optimal control function will have 2 components: $u(u_1, u_2)$.

2. THE SHIPS STABILIZATION IN ROLLING CASE

It is known that the maritime or spatial ships are disturbed in their movement (by the waves, wind or other mechanical parameters). The rolling oscillations that they are forced to do are represented in Figure 1 by the rotation angle $\theta(t)$, side by a fixed mark. To re-achieve the stable movement regime, these oscillations have to be damped, and this is done by the mashinery presented in Figure 1.



Fig. 1: A pump conected to two tanks

Basically, it consists in two tanks $(T_1 \text{ and } T_2)$ mounted simetrically of the longitudinal and vertical axis and a system of servo-mechanics and electronical leaded pumps (a hydraulic

regulator, R), which transfer fluid in the tanks, contra-ballancing the rolling oscillations. The hydraulic regulator is shown in figure 2.





In this mode of discharge or filling the tanks in contra ballance with the oscillation amplitude, a spell effect is achieved by the friction and the variation of the fluid mass (left - right), involving the asymptotic stability with intermittences. In the phase space, the spiral trajectories are wavy with the change of the movement sense, tending to the stable asymptotic focus.

The rolling angle θ (Figure 2) is signalized by the rolling indicator **4** in the gyroscopically quadrant

mounted on the ship. So, through the variation of this angle, the server **1** is involved by the pump (ex: in right side) an is opening the input of the fluid in the right half of the servo-motors **2**. This produce a displacement r of the bar **3**, proprtional with the angle θ .

Simplifying the model, the equation of the system with the free degree $\theta(t)$ is:

$$J\ddot{\theta} = -M\dot{\theta} - Kr$$
 (1)
where J is the inertia momentum of the ship,
relating the vertical axis which goes through the

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mass centre, $\theta(t)$ is the rolling angle side by the

is the linear momentum of $\theta = 0$, Mθ amortization by the friction of the viscous fluid, and r is the displacement of the mechanism arm owing to the rolling. I tis proportional with θ , so that $Kr = N\theta$. This is considered to be the the reaction momentum, necessary to damp the rolling oscillation, so that the ship returns to its normal course. To accelerate this amortization, the hydraulic absorber must keep the rolling angle in certain limits. In this way, this system is adding an "inverse link" (feedback) 5. The cylinder of the server acquires beside the rolling rotation (to the right) a double pressure. The server valves are closing and its operation is stopped. This is repeated acting to the left side, by discharging the right tank and filling the left one. So, a bigger amortization is achieved because of the momentum $\ensuremath{\,N\theta}$, and after a finite number of oscillations, the ship is stabilized to the null solution by decreasing the rolling oscillation amplitude (amortizing). The link is with "free bearing clearence" and for coulisse 6 an the 2B distance and the server body oscillating in the $\pm B$

space, this mean $r \pm B \approx \theta$, or $r = \theta \mp B$, with the "+" sign for $\dot{\theta} > 0$ and the -" sign for $\dot{\theta} < 0$

$$\ddot{\theta} + 2\lambda\dot{\theta} + \omega^2\theta = \pm\omega^2 B$$
 (2)

In the phase space, the study of the movement is carried aut in the following way: we are considering the movement right-left with $\theta \in (-B, B)$. Denoting $\theta = x_1 + B$, (2) becomes:

$$\dot{x}_1 + 2\lambda \dot{x}_1 + \omega^2 x_1 = 0, \delta = \lambda^2 - \omega^2 < 0$$
 (3)

For $n^2 < k^2$ we have a low resistance is linearly amortized side by $x \equiv 0$, having a displacing to the right face of the origin with B. he decrement of the oscillation is $\Delta = e^{-n\pi\omega}$. If the displacement is from left to right, then $\theta = x_2 - B$, resulting the following equation:

$$\ddot{x}_2 + 2\lambda \dot{x}_2 + \omega^2 x_2 = 0$$
 (4)

The amortization law is the same, but the oscillation will be displaced to the left with B. So, the trajectory in the phase space will trace a

logarithmic spiral (figure 3) in the (x, y) plane, $\dot{x} = y$ with the solution for the homogeneous case:

$$x = e^{-\lambda t} \left(C_1 \cos \delta t + C_2 \sin \delta t \right)$$
 (5)





Then, with $\theta = x \pm B$ for the non-homogeneous equation, the trajectory presented in figure 4 was obtained graphically: a cut was made on the x Ox

axis and the upper figure is displaced to the right with B, since the down figure is displaced to the left with B.



Fig. 4: The trajectory in the phase space in nonhomogeneous case

The spirals are merged by continuity and will ondulate to origin O thus: considering the initial position at t=0, $\dot{x}_0^i>0$, then with recurrence ton the left:

 $x_0^{i+1} = \left(x_0^i - B\right)\Delta - B = x_0^i \Delta - B(1+\Delta) \quad (6)$ The point will got o the left side of the origini if:

$$x_{0}^{i}\Delta - B(1+\Delta) > 0 \text{ or } x_{0}^{i} > B(1+\frac{1}{\Delta}) > 2B$$

(6) is conditioned by $\, 0 < x_0^i \leq B$.

If $B < x_0^i \le B(1 + \Delta^{-1})$, the trajectory falls to origin through the inferior side without cross in the interval (-B, B) – figure 2.

Recalling (1), we obtain (2):

$$\ddot{\theta} + 2\lambda\dot{\theta} + \omega^{2}\theta = \begin{cases} \omega^{2}B, \theta < 0 ; \theta \downarrow \\ -\omega^{2}B, \dot{\theta} > 0 ; \theta \uparrow \end{cases}$$

The resistance force is a discontinuous pulse (relay type): $F(\dot{\theta}) = 2\lambda \dot{\theta} \mp \omega^2 \theta$.

This equation caracterize an intermitent oscillation with a time-decreasing amplitude and the solutions will be recurrent, $\theta_n = \theta_n(t)$, where the initial conditions for $\theta_{n+1}(t)$ will be $\theta_{n+1}(t_n^0) = \theta_n(t_n^0)$ and $t^* = t_n^0$ will be the moment when $\dot{\theta}_n(t^*) = 0$. We will consider the first phase $\theta = \theta_1(t)$ when $\theta_1(t \equiv 0) = \theta_0$ and $\dot{\theta}_1(0) = 0$,, $\theta_0 > B$. The last condition is justified by the particular solutions $\theta = \pm B$ and taking into consideration that for $\theta_0 \in (-B, B)$ we have a ballanced regime, see figure 5.

$$\begin{aligned} \theta_1(t) &= B + \omega \frac{a_1}{\delta} e^{-\lambda t} \cos(\delta t - \phi) \\ \theta_{2n}(t) &= -B + \omega \frac{a_{2n}}{\delta} e^{-\lambda t} \cos(\delta t - \phi) \\ \theta_{2n-1}(t) &= B + \omega \frac{a_{2n-1}}{\delta} e^{-\lambda t} \cos(\delta t - \phi) \end{aligned}$$

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Fig. 5: Rolling amortization

3. OPTIMAL CONTROL IN ANGULAR SPEED STABILIZATION REGARDING FLUVIAL OR SPATIAL NAVES

We will consider a multi-engines ship with axialcylinder symmetry. Figure 6. We choose a system of axes $(Ox_1x_2x_3)$ as principal axes of inertia, where O is the masses, as a solid body rigidly fixed in point O. The vessel rotate with the angular velocity $\overline{\omega}(\omega_1, \omega_2, \omega_3)$, where ω_i are the rotation speeds around the axes Ox_i , $i = \overline{1,3}$; Ox_3 is the axis of longitudinal symmetry. Symmetrical with these axes the ship has turbojet propulsion generators $G_1(G_1^1, G_1^2)$ on Ox_1 with two nozzles, $G_2(G_2^1, G_2^2)$ on Ox_2 and generator G_3 on Ox_3 , with traction purpose. These reactive nozzles can create moments, being accompanied by gas dynamics wings, integrating gyroscopes, small jet shutters that can help to guide and stabilize the angular velocities regime. The controller [3], [5-7] is equipped with sensors and microprocessors for data processing and it may (with a rapid response) control the disrupted regime for optimal stabilization [12], [14]. Disturbances considered here may be due to turbines fuel, meteo external agents or environmental density. These naves may be rockets, spacecraft, capsule, modules, megadrones or submarines, torpedoes, etc [8], [12], [14].



Fig. 6: A multi-engine ship

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The angular velocities are $\omega_1 = x_1(t)$, $\omega_2 = x_2(t)$, $\omega_3 = x_3(t)$, the inertia momentums of the body are I₁, I₂, I₃ - simetrically (I₁ = I₂ = I), see figura 6.

We write the equations of angular velocities disturbed by external moments $M_i(t)$ [10], [12], [14]:

$$\begin{cases} I_{1}\dot{x}_{1} = (I_{2} - I_{3})x_{2}x_{3} + M_{1}(t) \\ I_{2}\dot{x}_{2} = (I_{3} - I_{1})x_{1}x_{3} + M_{2}(t) \\ I_{3}\dot{x}_{3} = (I_{1} - I_{2})x_{1}x_{2} + M_{3}(t) \end{cases}$$
(1)

We suppose that moments $M_i(t)$ are caused by propelling forces $G(g_1, g_2, g_3)$:

$$\begin{array}{l} h_1 = l \cdot g_1(t), \ h_2 = l \cdot g_2(t), \ h_3 = l \cdot g_3(t) \ \mbox{(2)} \\ \mbox{And} \quad taking \quad into \quad consideration \quad the \quad simetry \\ I_1 = I_2 := I \ , \ we \ have: \end{array}$$

$$\begin{cases} \dot{x}_{1} = \frac{I - I_{3}}{I} x_{2} x_{3} + \frac{1}{I} g_{1}(t) \\ \dot{x}_{2} = -\frac{I - I_{2}}{I} x_{3} x_{1} + \frac{1}{I} g_{1}(t) \\ \dot{x}_{3} = \frac{h}{I_{3}} g_{3}(t) \end{cases}$$
(3)

System (1) with $M_i(t)=0$ is in stable equilibrium around O(0,0,0) (undisturbed).

Let's suppose that at $t_0 = 0$ we have the disturbed position

 $x_1(0) = \alpha_1, x_2(0) = \alpha_2, x_3(0) = \alpha_3$ (4) If the traction $g_3(t)$ is known, than we have

$$x_3(t) = \alpha_3 + \int_0^t \frac{h}{I_3} g_3(\tau) d\tau$$
(5)

This shows that x_3 can be controlled independently of x_1 and x_2 , but x_3 can influence in (3) variables x_1 and x_2 . We suppose that the rapid reaction response time is short and x_3 may be considered constant $x_3 = \alpha_3$, so $g_3 \cong 0$. We also suppose that forces g_1 , g_2 are bounded

$$\mathbf{g}_1(\mathbf{t}) \leq \mathbf{L}$$
, $|\mathbf{g}_2(\mathbf{t})| \leq \mathbf{L}$ (6)

In this case the system (3) is linearized and we introduce the control function $u(u_1(t), u_2(t))$ to control optimum stabilization of disturbed solution to O (x₁=0, x₂=0) for system (7) in minimal time.

$$\begin{cases} \dot{\mathbf{x}}_1 = \omega \mathbf{x}_2 + \mathbf{k} \mathbf{u}_1(\mathbf{t}) \equiv \mathbf{f}_1 \\ \dot{\mathbf{x}}_2 = -\omega \mathbf{x}_1 + \mathbf{k} \mathbf{u}_2(\mathbf{t}) \equiv \mathbf{f}_2 \end{cases}$$
(7)

where:

$$\begin{cases} \omega = \frac{I - I_3}{I} \alpha_3; & \omega > 0 \\ k = \frac{1}{I}L; \ k > 0 \\ u_i(t) = \frac{g_i(t)}{L}; & i = 1, 2 \end{cases}$$
(7')

We note that state equations (7) in the phase space (x_1Ox_2) , $X(x_1(t);x_2(t))$, with control parameters $(u_1(t);u_2(t))$ satisfy the conditions:

$$\begin{cases} -1 \le u_1(t) \le 1\\ -1 \le u_2(t) \le 1; \end{cases} \quad U := [-1, 1]$$
(8)

and hence the allowable plan U (u_1Ou_2) is a compact square.

The technical sense in equations (7) for U_i (t) is to find forces $g_i(t) \leftrightarrow u_i(t)$ to reduce speeds $(x_1, x_2) \rightarrow (0, 0)$ on optimal paths starting from $M_0(\alpha_1, \alpha_2)$ at time t_0 to reach the final target O (0, 0) at the time $t_f > t_0$ so the transfer time to be minimal (o.c.p.) [2], [3], [9], [7].

4. MINIMAL TIME CRITERION. EXTREMUM PRINCIPLE

Let's consider a system described by the state equations

 $\dot{\mathbf{x}}_{i} = \mathbf{f}_{i}(\mathbf{t}, \mathbf{x}(\mathbf{t}), \mathbf{u}(\mathbf{t})), \mathbf{t} \in [\mathbf{t}_{0}, \mathbf{t}_{1}] \subseteq \mathbf{R}_{+}, i = \overline{\mathbf{1}, \mathbf{n}}$ (9)

 $\begin{array}{ll} \mbox{where} & \mbox{the state function is} \\ x(t) \!=\! \begin{pmatrix} x_1\,, x_2\,, ..., x_n\, \end{pmatrix}\!, x_i \in \! X \!\subset\! {{\hbox{\bf R}}}^n \mbox{ and the control} \\ \mbox{function} & u(t) \!=\! \begin{pmatrix} u_1\,, u_2\,, ..., u_m\, \end{pmatrix}\! \subset U \!\subset\! {{\hbox{\bf R}}}^m \mbox{ with} \\ m \leq n \,. \end{array}$

Functions fi meet the regularity conditions and U is the allowable domain of parameters $u_i(t)$. System (9) respects the given initial conditions (I):

(I)
$$x_i(t_0) = x_i^0 \in R, i = \overline{1, n}$$
 (10)

determining the disturbed initial state $X^0 \in S^0$, where S^0 is the variety on which X^0 is fixed. Assuming that Cauchy problem (9) (10) has $x_i = x_i \left(t, t_0, x^0, u\right), t \geq t_0$ as unique solution, we request that this trajectory transfer the system in the state $X^1 \in S^1$, where X^1 is fixed on S^1 (target (final) state - usually steady state in the final moment $t_1 = t_f$ horizon pool) - $t_0 \leq t_1$:

(F)
$$x_i(t_1) = x_i^1 \in \mathbb{R}$$
, $i = 1, n$ (11)
The final moment t_1 will be determined using
"minimal time criterion" - rapid response
 $\min_{u \in U} (t_1 - t_0) = t^*$, extremizing the performance

index J = J [u] [1], [2], [9].Let's consider the index functional, see [1], [9], [10]:

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$$J(u) = \int_{t_0}^{t_1} f_0(x, u, t) dt$$
 (12)

where f_0 is a characteristic function (Lagrangean) with:

 $\dot{\mathbf{x}}_{0}(t) \equiv \mathbf{f}_{0}(t, \mathbf{x}, \mathbf{u}); \mathbf{x}_{0}(t_{0}) = \mathbf{C}_{0}$ (13)

The optimal control problem (P.C.O.) is to determine an optimal admissible command $u^* \in U$ to extremize (12) so that the original system (9) (10) (I) is transferred to the final system (11) (F) in minimal time (minimum criteria). "Extreme Pontriaguine principle" (P.E.) will be calling to solve it. We auxiliary introduce the multipliers $\lambda(t) = (\lambda_0(t), \lambda_1(t), ..., \lambda_n(t))$ as non-null solution of the adjunct system [1] [9] [7] [10]:

$$\dot{\lambda}_{i} = -\sum_{j=0}^{n} \left(\frac{\partial f_{j}}{\partial x_{i}} \right)_{0} \lambda_{i} ; \lambda_{i}(t_{0}) = c_{i}, i = \overline{1, n}$$
(14)

asociated with (9)...(13) with arbitrary constants c_i (but notg all of them arbitarary), wich will finally become the controller parameters.

We note that (14), if it's linearized: $\dot{X} = AX + BU$ i

of the final null position O, $A = (a_{ij}) = \left(\frac{\partial f_i}{\partial x_j}\right)_0$ i.e.

 $\dot{\lambda}=-A'\lambda$, where $\dot{\lambda}=-A'\lambda\,,~A'$ is the transposed matrix.

We consider the lagrangean like f_0 and (12) :

 $L(t, x(t), u(t)) \equiv f_0(x, u, t) \equiv 1, \dot{x}_0 = f_0 \equiv 1$ (15)

$$J(u) = \int_{t_0}^{1} Ldt = t_1 - t_0$$
min $J(u) = J(u^*) = \min(t_0 - t_0) = t^*$
(16)

 $\min_{u} J(u) = J(u) = \min_{u} (t_1 - t_0) = t$ We build the generalized Hamiltonian [9] [10] associated with (11) (12) (13) (14):

$$H(t, x(t), \lambda(t), u(t)) = \lambda_0 \dot{x}_0 + \sum_{i=1}^n \lambda_i \dot{x}_i \quad (17)$$

where $\dot{x}_0 = f_0 \equiv 1$, with $\dot{\lambda}_0 = \left(\frac{\partial f_0}{\partial x}\right)_0 \lambda \equiv 0$, $\lambda_0 \equiv C_0$

From (11) and (17) we have:

$$H \equiv \lambda_0 + \sum_{i=1}^{n} \lambda_i f_i(t, x, u)$$
(18)

and we built the canonic attached and adjunct system [1] [2] [7] [9]:

$$\dot{x}_{i} = \frac{\partial H}{\partial \lambda_{i}}$$
(19)

$$\dot{\lambda}_{i} = -\frac{\partial H}{\partial x_{i}}, i = \overline{1, n}, t \in [t_{0}, t_{1}]$$
 (20)

with $u_i \in [-1,1]$ and initial conditions $x_i(t_0) = x_i^0$, $\lambda_i(t_0) = c_i$ This system is:

 $\dot{\mathbf{x}} = \mathbf{f} (\mathbf{t} \mathbf{x} \mathbf{u}) \dot{\mathbf{x}} = \lambda \sum_{j=1}^{n} \left(\partial \mathbf{f}_{j} \right)$

$$\dot{\mathbf{x}}_{i} = \mathbf{f}_{i}(\mathbf{t}, \mathbf{x}, \mathbf{u}); \dot{\boldsymbol{\lambda}}_{i} = -\lambda_{i} \sum_{j=1}^{\infty} \left(\frac{\partial \mathbf{1}_{j}}{\partial \mathbf{x}_{i}} \right)_{0}$$
(21)

with 2n unknowns: x_i , λ_i şi 2n conditions, where u* is previously and optimal choosed.

"Pontreaguine minimum principle" theorem ([1] [2] [9]): A necessary condition for the existence of an optimal solution $u^* \in U$ $(x_i \in [-1,1])$ minimizing (11), $\min_u J(u) = J(u^*) \equiv \min_u (t_1 - t_0) = t^*$ associated with (9)(10)(11)(18), where $\min_u H(t, x, \lambda, u) = H^*(t, x, \lambda, u^*) = 0, \forall (t, x, \lambda)$ is that trajectories x_i , λ_i respect (19)(20)(21) $\forall t \in [t_0, t^*]$ with: $H(t, x, \lambda, u) \ge H^*(t, x, \lambda, u^*) \ge 0,$ $\min(H) = H^*(t^*, x^*, \lambda^*, u^*, C) = 0$ (22)

Remarks:

1) We may take $\lambda_0 = c_0 \equiv 1$ and H(t) is minimized determining the vector $\lambda = (\lambda_1, \lambda_2, ..., \lambda_n)$ so that the speed $\dot{X} = (\dot{x}_i)$ projection on λ vector to be

minimum: $\min\left(\sum_{i=1}^n \lambda_i \dot{x}_i\right)$.

2) After builting H, (17)(18) with H = H(u), we fiind u* with $-1 \le u_i \le 1$, generally with $\frac{\partial H}{\partial u} = 0$; but, if H is linear ieste liniar in u, H = a+bu_1+cu_2, then, according to linear programming with $H \ge 0$ in compact square $u(t) \in U$ included in (H, u1, u2) space, the minimum H(u*)=0 will be in the square tips $u^* = \{ u_1^* = -1, u_2^* = 1 \}, (u_1^* = 1, u_2^* = -1) \};$ the solutions will be $x = x(t, u^*), \lambda = \lambda(t, u^*).$

5. ANGULAR SPEEDS STABILIZATION OPTIMAL CONTROL

We still apply the algorithm (9) - (22) To (1) – (8); and build Hamiltonian (17) (18) associated with the system (7):

$$H = 1 + \lambda_1 x_1 + \lambda_2 x_2 =$$

= $1 + \lambda_1 \omega x_2 - \lambda_2 \omega x_1 + \lambda_1 k x_1 + \lambda_2 k x_2 \ge 0$ (23)

We note that $H = H(u_1, u_2) = a + bu_1 + cu_2$ is linear and positive in the compact square $u_1(t) \in [-1;1]$, $u_2(t) \in [-1;1]$; so in the space (H, u_1 , u_2) H has a null minimum in the sqare tips: $H \ge H_{\min} = H(u_1^*, u_2^*) = 0$

So, if
$$u_1^* = -sgn(\lambda_1); u_2^* = -sgn(\lambda_2)$$
 (24)

the trajectories $C_1^{\pm}(u_1 = 1, u_2 = -1)$ or $C_1^{\mp}(u_1 = -1, u_2 = 1)$ will be obtained, and they will tend to origin $O(x_1 = 0, x_2 = 0)$.

The systemul is autonom and $\overset{\cdot}{u}$ is pulse-type. It Its that the controller will be a relay-type one, acting with or without commutation [1] [2] [7]. We solve the canonic system (13)(14), effectiv (7) with initial conditions $x_i(t_0=0)=\alpha_i$, i=1,2 and $u\in U$:

$$\begin{cases} \omega x_1 - u_2^* k = (\omega \alpha_1 - u_2^* k) \cos \omega t + (\omega \alpha_2 + u_1^* k) \sin \omega t \\ \omega x_2 + u_1^* k = -(\omega \alpha_1 - u_2^* k) \sin \omega t + (\omega \alpha_2 + u_1^* k) \cos \omega t \end{cases}$$
(25)

$$\left(x_1 - \frac{u_2^*k}{\omega}\right)^2 + \left(x_2 + \frac{u_1^*k}{\omega}\right)^2 = \left(\alpha_1 - \frac{u_2^*k}{\omega}\right)^2 + \left(\alpha_2 + \frac{u_1^*k}{\omega}\right)^2$$
(26)

We note that these trajectories are circles with

centers $O^*\left(\frac{u_2^*k}{\omega}, -\frac{u_1^*k}{\omega}\right)$ and radius $R^2 = \left(\alpha_1 - \frac{u_2^*k}{\omega}\right)^2 + \left(\alpha_2 + \frac{u_1^*k}{\omega}\right)^2$, but they are not

reaching thr target point O(0,0) for $\forall (\alpha_1, \alpha_2)$. If in (26) $x_1 \equiv 0, x_2 \equiv 0$, we will get for $(\alpha) = (x)$ the compatible circles – a bilocal problem.

$$\begin{pmatrix} \omega x_1 - u_2^* k \end{pmatrix}^2 + \left(\omega x_2 + u_1^* k \right)^2 = 2k^2; R = \frac{k\sqrt{2}}{\omega};$$

$$O_0^* \left(\frac{u_2^* k}{\omega}, -\frac{u_1^* k}{\omega} \right)$$

$$\begin{cases} x_1 - \frac{u_2^* k}{\omega} = \frac{k\sqrt{2}}{\omega} \cos\theta \\ x_2 + \frac{u_1^* k}{\omega} = \frac{k\sqrt{2}}{\omega} \sin\theta \end{cases}; \theta = \theta_0 + \omega (t - t_0)$$
(28)

We note that the origin is on the circles from this family and their centers are on the first bisecting line (Oz₁) of the system x₁Ox₂. lor on the second one (Oz₂), with the (z₁Oz₂) axis system, see Figure 7a. Choosing the optimal circles depends on the optimal control u* so that $H^*(u^*) \ge 0$; if the

X system would be rotated with
$$\frac{\pi}{4}$$
:

 $z \rightarrow z = x \cdot e^{i\frac{\pi}{4}}$, we note that from the $H(u^*) \ge 0$ pozitivity with (23) and (24) it results the trajectory

on the upper (towards Oz₁) halfcircles $\{C_1^*\}$ from the first quadrant and the lower halfcircles, $\{C_2^*\}$ from the third quadrant, i.e: $\{C_{10}^{\mp}:u_1^*=-1,u_2^*=1\}\cup\{C_{20}^{\pm}:u_1^*=1,u_2^*=-1\}$, with centers respectively: $O_{10}^{\mp}\left(\frac{k}{\omega},\frac{k}{\omega}\right)$ and

$$\begin{split} &O_{20}^{\pm}\!\!\left(-\frac{k}{\omega},\!-\frac{k}{\omega}\right)\!\!. \quad \text{Choosing the initial point} \\ &M^0\!\left(\!\alpha_1^0,\!\alpha_2^0\right)\!\!\in\!C_{10}^{\mp} \text{ in the moment } t_0, \text{ corresponds to} \end{split}$$

the angle θ_0 towards Ox_1 : $\tan \theta_0 = \frac{\alpha_2^0 - \frac{k}{\omega}}{\alpha_1^0 - \frac{k}{\omega}}$;

It may be observed on figure 7a that $\theta_0 \in \left[\frac{\pi}{4}, \frac{5\pi}{4}\right]$, with $\theta = \omega t + \frac{\pi}{4}$.

$$\begin{aligned} \theta_0 &\in \left[\frac{\pi}{4}, \frac{5\pi}{4}\right], t_0 \in \left[0, \frac{\pi}{\omega}\right], \\ t^* &= \frac{1}{\omega} \left(\frac{5\pi}{4} - \theta_0\right), \alpha_1^0 > 0, \alpha_2^0 > 0 \end{aligned}$$
 (29)

Analog and asimetricaly for the circle C_2^{\pm} . **Remarck.** The optimal trajectories are periodical $T = \frac{2\pi}{\omega}$; the halfcircles $\left\{C_{1j}^{\mp}\right\} \cup \left\{C_{2j}^{\pm}\right\}$ are tangent, with centers

$$O_{1j}^{\mp}\left((2j+1)\frac{k}{\omega},(2j+1)\frac{k}{\omega}\right), O_{2j}^{\pm}\left(-(2j+1)\frac{k}{\omega},-(2j+1)\frac{k}{\omega}\right), j=0,1,2,...$$

and radius $R=\frac{k}{\omega}\,;\,\,$ for example, if the starting point is $M_{0j}\in C_{1j}^{\mp}$, then the minimal time will be $t^{*}=j\frac{k}{\omega}+\frac{1}{\omega}\biggl[\frac{5\pi}{4}-\theta_{0}\Bigl(M_{0j}\Bigr)\biggr],\,\,\,$ without relay commutation.

There are situations when the starting point is not on the small halfcircles, for example $P_0(\gamma_1,\gamma_2)$ in the third quadrant in figure 7b. In this case we choose from C_{2j}^{\pm} a halfcircle O_{2j}^{\pm} with radius R_{Γ} denoted Γ^{\pm} . This circle crosses one of the halfcircles C_{1j}^{\mp} . This is the commutation moment $\left(u_i \rightarrow -u_i, i=1,2\right)$, the new trajectory starting from the starting point and ending in origin.

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Fig. 7: Optimal trejectories

a) With no relay commutation b) with relay commutation

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