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CALCULUS AND MATERIALS FOR STIRLING ENGINE'S BOLTER AND REGENATOR

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Abstract: The performances of the Stirling engine are affected by the convection coefficient and the "X" factor and not only by the variation of the gas quantity from the cylinder with the medium pressure variation. The convection factor indicates that a sensibility study concerning the characteristic parameters is mandatory.

Keywords: Stirling, cycle, engine, convection.

1. INTRODUCTION

We will consider the most common case, which presents the regenerator as a "pressed bolters package". In figure 1 are presented the geometrical characteristics for the

We can assimilate regenerator's area with a square area equivalent to the regenerator's:

$$A_R = \frac{\pi D_R^2}{4} = a^2 \tag{1}$$

where D_R – regenerator's diameter;

$$a = D_R \sqrt{\frac{\pi}{4}} = \frac{D_R \cdot \sqrt{\pi}}{2} \tag{2}$$

If L is regenerator's length, then:

$$N_s = \frac{L}{2d} \tag{3}$$

- N_s - total bolter number;

- d – bolter's wire diameter

From here results:

(wires in the equivalent square in X direction) =
$$\frac{a}{b+d}$$
 (4)

(bolter`s wires in both directions) =
$$\frac{2a}{b+d}$$
 (5)

where b is the distance between two bolter's wires

(bolter`s wires lenght) =
$$\frac{2a^2}{b+d}$$
 (6)

(regenerator`s wires lenght) =

(bolters number)
$$\cdot$$
 (bolter's wires length) = L_f (7)

$$L_f = \frac{2a^2}{b+d} \cdot \frac{L}{2d} = \frac{L \cdot a^2}{d(b+d)} \tag{8}$$

With these relations we can determine the radius AR and the weight m_R . Knowing a form of second relation and using it in the 8th relation, results:

$$L_f = \frac{L \cdot \frac{\pi D_R^2}{4}}{d(b+d)} = \frac{\pi D_R^2 \cdot L}{4d(b+d)}$$
 The regenerator's area will be:

$$A_R = \pi \cdot d \cdot L_f = \frac{\pi^2 \cdot D_R^2 \cdot L \cdot d}{4(b+d) \cdot d} \tag{10}$$

meaning:

$$A_R = \frac{\pi^2 \cdot D_R^2 \cdot L}{4(b+d)}$$
 The regenerator's weight will be:

$$m_R = L_f \cdot A_f \cdot \rho_{metal} = \frac{\pi D_R^2 L}{4d(b+d)} \cdot \frac{\pi d^2}{4} \rho_{metal}$$
 (12)

- A_f – wire's area;

- ρ_{metal}=ρ_m – metal's density

$$m_R = \frac{\pi^2 \cdot D_R^2 \cdot d \cdot \rho_m \cdot L}{16(b+d)} \tag{13}$$

The ratio $\frac{m_R}{A_P}$ will be:

$$\frac{m_R}{A_R} = \frac{\pi^2 \cdot D_R^2 \cdot d \cdot \rho_m \cdot L}{16(b+d)} \cdot \frac{4(b+d)}{\pi^2 \cdot D_R^2 \cdot L} = \frac{d \cdot \rho_m}{4}$$
(14)

The relation required to determine the parameter, according with the regenerator's properties will be:

$$X = 1 - \frac{\frac{\pi^2 \cdot D_R^2 \cdot d \cdot \rho_m \cdot L \cdot c_R}{16(b+d)m_g \cdot c_{v_g}}}{\frac{n_r \cdot d \cdot \rho_m \cdot c_R}{15\alpha} - 1}$$
(15)

In this relation, number 15 is used because the sinusoidal movement is resulted from a quarter rotation,

meaning
$$\frac{60}{4} = 15$$
.

For accomplishing the operational status for the 15th relation the following two conditions are required:

- determination of the regenerator's convection coefficient;
- bordering the X factor in general optimization scheme for Stirling engine.

2. THE REGENERATOR'S CONVECTION COEFFICIENT **EVALUATION**

The following evaluation is based upon the description of the relation between the similitude criteria for the regenerator indicated by Organ [Thermodynamics and Gas dynamics of Stirling Cycle Machine, pag 113]:

$$N_{ST} \cdot N_{Pr}^{\frac{2}{3}} = \frac{1,25}{\sqrt{N_{Re}}}$$
 (16)

According to the Romanian scientific standards the relation will be written as:

$$St \cdot \Pr^{\frac{2}{3}} = \frac{1,25}{\sqrt{\text{Re}_D}}$$
 (17)

relation used for a pore - ratio with following values:

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$$\varepsilon_p = 0.602 \div 0.832$$
 (18)

where ϵ_{p} is the porosity.

Incopera indicates a similar relation [1, page 292]:

$$St \cdot \Pr^{\frac{2}{3}} = \frac{0.79}{\varepsilon_p \cdot \operatorname{Re}^{0.576}} \tag{19}$$

The two formulas would be almost identical if the porosity formula, used in the second relation would be

$$\varepsilon_p = \frac{0.79}{1.25} = 0.63$$
 , included in the first formula domain.

We will consider the Incopera formula first, continuing with the Organ formula, because a comparative study would be valuable in order to assign the possible implications for Stirling engines:

$$\operatorname{Re}_{D} = \frac{D \cdot \rho \cdot \overline{w}}{\mu} = \frac{D_{R} \cdot \overline{w}}{v} \tag{20}$$

where:

- $\overline{\boldsymbol{W}}$ is the medium speed., μ dynamic viscosity, υ kinematic viscosity.

The coefficient Re_D is determined for medium speed \overline{W} and D_R, the diameter for regenerator's empty shell.

$$St = \frac{\alpha}{\rho \cdot \overline{W} \cdot c_p} \tag{21}$$

and

$$\Pr = \frac{v}{a} = \frac{v}{\lambda/(\rho \cdot c_p)} = \frac{\rho \cdot c_p \cdot v}{\lambda}$$
 (22)

$$\varepsilon_{p} = \frac{V_{total, \text{Reg}} - V_{fire, site}}{V_{total, \text{Reg}}} = 1 - \frac{V_{fire, site}}{V_{total, \text{Reg}}}$$
(23)

with V_{fire.site} - the volume of bolter's wire.

Developing the presented relations, we can obtain:

$$\varepsilon_p = 1 - \frac{\pi \cdot d}{4(b+d)} \tag{24}$$

Using equation number 21 in 9 and 23 statements results:

$$\frac{\alpha}{\rho \cdot \overline{w} \cdot c_p} \cdot \Pr^{\frac{2}{3}} = \frac{0.79}{\left[1 - \frac{\pi d}{4(b+d)}\right] \cdot \operatorname{Re}^{0.576}}$$
(25)

This relation makes the determination of the α coefficient depending of Pr and Re criterions:

$$\alpha = \frac{0.79 \rho \overline{w} c_p}{\left[1 - \frac{\pi d}{4(b+d)}\right] \cdot \Pr^{\frac{2}{3}} \cdot \operatorname{Re}^{0.576}}$$
 (26)

The 26th relation may be developed as following:

$$\alpha = \frac{0.79 \, \rho \cdot \overline{w} \cdot c_p}{\left[1 - \frac{\pi \cdot d}{4(b+d)}\right] \cdot \left(\frac{\rho \cdot c_p \cdot v}{\lambda}\right)^{\frac{2}{3}} \cdot \left(\frac{D_R \cdot \overline{w}}{v}\right)^{0.576}}$$
(27)

equivalent with:

$$\alpha = \frac{0.79 \rho^{\frac{1}{3}} \cdot \overline{w}^{0.424} \cdot c_p^{\frac{1}{3}} \cdot \lambda^{\frac{2}{3}}}{\left[1 - \frac{\pi \cdot d}{4(b+d)}\right] \cdot v^{0.09} \cdot D_R^{0.576}}$$
(28)

3. REFERENCES

- [1] Florea, T., Florea, E.: Studiul ireversibilitătii proceselor termodinamice ale pompei de căldură cu vapori, Conferinta Termoenergeticienilor, Brasov, 1989.
- [2] Bejan, A.: *Theory of Heat Transfer-Irreversible Power Plants*, Int. J. Heat Mass Transfer, vol. 31, 1988.
- [3] Organ, J.A.: Thermodynamic Design of Stirling Cycle Machine, Proc. Instn. Mech. Engrs., 1987.
- [4] Heywood, J.B.: Internal Combustion Engine Fundamentals, McGraw-Hill Book Company, New York, 1988.
- [5] Organ, J.A.: Thermodynamics and Gas Dynamics of Stirling Cycle Machine, Cambridge University Press, Cambridge, 1992.
- [6] Qvale, E.B., Smith, J.L.: A Mathematical Model for Steady Operation of Stirling-Type Engines, Journal of Engineering for Power, 1968.
- [7] Incropera, F., De Witt, D.: Introduction to Heat Transfer, John Wiley&Sons, New York, 1996.
- [8] Florea, T., Florea, E.: Determinarea sursei de entropie într-un sistem continuu, Conferinta Termoenergeticienilor, Brasov, 1986.
- [9] Petrescu, S., Costea, M., Florea, T., Harman, C.: Determination of the Losses in a Stirling Cycle through Use of a pV/px Diagram, paper submitted and accepted for presentation and publication to the International Conference of Efficiency, Costs, Optimization, Simulation and Environmental Aspects of Energy and Process Systems, ECOS'2000, Entschede, Netherlands, July 5-7, 2000.
- [10] Petrescu, S., Harman, C., Costea, M., Florea, T.: A Method for Determination of the Performances of Stirling Machines Based on a pV/px Diagram and First Law for Processes with Finite Speed, paper submitted and accepted for presentation and publication to The 5th World Conference on Integrated Design & Process Technology, Dallas, Texas, USA, June 4-8, 2000.
- [11] Petrescu, S., Florea, T., Harman, C., Costea, M.: A Method for Calculating the Coefficient for the Regenerative Losses in Stirling Machines, Paper submitted and accepted for presentation and publication to the European Stirling Forum 2000, Oswabrück, Germany, February 22-24, 2000.
- [12] Petrescu, S., Florea, T., Costea, M., Harman, C.: Application of the Direct Method to Irreversible Stirling Cycles with Finite Speed, International Journal of Energy Research, USA, nov. 2000.
- [13] Petrescu, S., Florea, T., Harman, C., Costea, M.: A New Technique for Determining the Coefficient of Regenerative Losses in Stirling Machines, ECOS 2001, Istanbul, Turkey, 4-6 July, 2001.